

烃类无相变多股流换热器加工



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摘 要 气体分离轻烃加工往往遇到多股流换热,且多属一股原料气对多股分离产品气体换热。本文分析在不发生相变传热时,所涉及的热物性、结构、传热计算的物理模型、计算方法与实用软件。

主题词 轻烃 加工 气流 换热器 物理性质 结构设计 传热 计算

在轻烃加工、气体分离中,为使原料气冷却、产品气复热,工艺流程多涉及原料气与产品气之间的多股流换热。利用板式换热器积木式结构特点,组织多股流体在同一个换热器内进行换热,这不仅可以简化流程、管网,而且在合理布置换热通道时,尚可收到高效、紧凑与降低投资运行费用的功效。故多股流板式换热器在石油化工、天然气液化与分离、低温工程中获得了广泛应用。这种换热器相对于两股流常规换热,具有复杂得多的温度场,设计方法亦颇具特点。本文就多股流、无相变传热所涉及的物理模型、结构设计、传热计算、设计软件进行分析。至于相变多股流传热将另文论述(本刊1996年第2期将刊出)。

热物性计算

1. 密度

以真实气体状态方程 SHBWR^[1] 确定密度,其形式为:

$$\begin{aligned} p = & \rho RT + \left(B_0 RT - A_0 - \frac{C_0}{T^2} + \frac{D_0}{T^3} - \frac{E_0}{T^4} \right) \rho^2 \\ & + \left(b RT - a - \frac{d}{T} \right) \rho^3 + a \left(a + \frac{d}{T} \right) \rho^6 \\ & + \frac{c \rho^3}{T^2} (1 + r \rho^2) \exp(-r \rho^2) \end{aligned} \quad (1)$$

纯组分参数 $B_{0i}, A_{0i}, \dots, r_i$ 为临界参数 T_{ci}, ρ_{ci} , 偏心因子 ω_i 的函数,其关联式如下:

$$\rho_{ci} B_{0i} = A_1 + B_1 \omega_i \quad (2)$$

500 h 的试验证明其结构是合理的。

整机性能测试和可靠试验

整机性能测试和可靠试验都是以空气为介质,进气压力 p_i 为 0.3 MPa,排气压力 p_a 为 4.5 MPa,吸气状态下气量 V_m 为 9.5 m³/min,压缩机轴功率 K 为 245 kW 进行的,性能测试结果表明全部达到设计要求,其性能参数见表 4。在连续 500 h 的可靠性试验中,机组未出现任何故障,试验结束后对机组进行了全面的检测,各运动部位磨损均匀且较小。

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表 4 产品性能参数表

Table 4. Performance parameters of the present product

发 动 机		压 缩 机	
型 式	双缸二冲程 电点火	型 式	双列双作用 往复活塞式
燃料气种类	低含硫天然气	压缩介质	天然气
有效功率 (kW)	268	排气量 (m ³ /min)	6~17.5
转 速 (r/min)	400	吸气压力 (MPa)	0.05~1.3
最低稳定转速 (r/min)	250	排气压力 (MPa)	2~4.5
燃料有效 消耗率 [m ³ /(kW·h)]	0.33	吸气温度 (°C)	20
气缸直径 (mm)	Ø380	排气冷却后 温度(°C)	46
冲 程 (mm)	406.4	气缸直径 (mm)	Ø330/Ø180
平均有效压力 (MPa)	0.443	活塞行程 (mm)	280
机组总重量 (t)	22.6	最大活塞力 (kN)	120

$$\frac{\rho_{ci} A_{ci}}{R T_{ci}} = A_2 + B_2 \omega_i \quad (3)$$

$$\frac{\rho_{ci} C_{ci}}{R T_{ci}^3} = A_3 + B_3 \omega_i \quad (4)$$

$$\rho_{ci}^2 r_i = A_4 + B_4 \omega_i \quad (5)$$

$$\rho_{ci}^2 b_i = A_5 + B_5 \omega_i \quad (6)$$

$$\frac{\rho_{ci}^2 a_i}{R T_{ci}} = A_6 + B_6 \omega_i \quad (7)$$

$$\rho_{ci}^3 a_i = A_7 + B_7 \omega_i \quad (8)$$

$$\frac{\rho_{ci}^2 C_i}{R T_{ci}^3} = A_8 + B_8 \omega_i \quad (9)$$

$$\frac{\rho_{ci} D_i}{R T_{ci}^4} = A_9 + B_9 \omega_i \quad (10)$$

$$\frac{\rho_{ci} d_i}{R T_{ci}^2} = A_{10} + B_{10} \omega_i \quad (11)$$

$$\frac{\rho_{ci} E_{ci}}{R T_{ci}^5} = A_{11} + B_{11} \omega_i \exp(-3.8\omega_i) \quad (12)$$

通用常数 $A_1, A_2, \dots, A_{11}$ 见表 1。

表 1 通用常数

Table 1. General constants

j	A_j	B_j	j	A_j	B_j
1	0.443 690	0.115 449	7	0.070 523 3	-0.044 448
2	1.284 380	-0.920 731	8	0.504 087	1.322 45
3	0.356 306	1.708 71	9	0.030 745 2	0.179 433
4	0.544 979	-0.270 896	10	0.073 282 8	0.463 492
5	0.528 629	0.349 261	11	0.006 450	-0.022 143
6	0.484 011	0.754 130			

对混合工质,其混合规则按:

$$B_o = \sum_i X_i B_{oi} \quad (13)$$

$$A_o = \sum_i \sum_j X_i X_j A_{oi}^{1/2} A_{oj}^{1/2} (1 - K_{ij}) \quad (14)$$

$$C_o = \sum_i \sum_j X_i X_j C_{oi}^{1/2} C_{oj}^{1/2} (1 - K_{ij})^3 \quad (15)$$

$$r = \left(\sum_i X_i r_i^{1/2} \right)^2 \quad (16)$$

$$b = \left(\sum_i X_i b_i^{1/2} \right)^3 \quad (17)$$

$$a = \left(\sum_i X_i a_i^{1/2} \right)^3 \quad (18)$$

$$\alpha = \left(\sum_i X_i \alpha_i^{1/2} \right)^3 \quad (19)$$

$$c = \left(\sum_i X_i c_i^{1/2} \right)^3 \quad (20)$$

$$D_o = \sum_i \sum_j X_i X_j D_{oi}^{1/2} D_{oj}^{1/2} (1 - K_{ij})^4 \quad (21)$$

$$d = \left(\sum_i X_i d_i^{1/2} \right)^3 \quad (22)$$

$$E_o = \sum_i \sum_j X_i X_j E_{oi}^{1/2} E_{oj}^{1/2} (1 - K_{ij})^5 \quad (23)$$

由 SHBWR 方程计算密度不能直接求解,需要

迭代,真实密度根的求解是个关键,详见文献[2]。

2. 比热容

理想气体纯组分比热容:

$$C_p^0 = A + BT + CT^2 + DT^3 \quad (24)$$

理想气体混合物比热容:

$$C_{pm}^0 = \sum_j Y_j C_{pj}^0 \quad (25)$$

实际气体比热容:

$$C_p = C_p^0 + \Delta C_p \quad (26)$$

按 L-K 方程比热容差:

$$\Delta C_p = (\Delta C_p)^{(0)} + \omega (\Delta C_p)^{(1)} = (C_p - C_p^0)^{(0)} + \frac{\omega}{\omega^{(R)}} [(C_p - C_p^0)^{(R)} (C_p - C_p^0)^{(0)}] \quad (27)$$

式 (27) 中热容差分项 $\left(\frac{C_p - C_p^0}{R} \right)^{(0)}$ 为对比压力 p_r , 对比温度 T_r 的函数, 列表见文献[3]。本软件系用插值数值算法。

3. 粘度

低压非极性气体:

$$\mu \xi = 4.610 T_r^{0.618} - 2.04 \exp(-0.449 T_r) + 1.94 \exp(-4.058 T_r) + 0.1 \quad (28)$$

低压极性气体氢键类, $T_r < 2.0$ 时:

$$\mu \xi = (0.755 T_r - 0.055) Z_c^{-1.25} \quad (29)$$

低压极性气体非氢键类, $T_r < 2.5$ 时:

$$\mu \xi = (1.9 T_r - 0.29)^{0.8} Z_c^{-1/2} \quad (30)$$

式中, $\xi = T_c^{1/6} M^{-1/2} p_c^{-1/2}$

低压气体混合物粘度:

$$\mu_m = \sum_i \frac{Y_i \mu_i}{\sum_j Y_j \varphi_j} \quad (31)$$

相互作用系数 φ_j , 由 Sutherland 关联式:

$$\varphi_j = \epsilon_{ij} \frac{1}{4} \left(\frac{M_{ij}}{M_j} \right)^{-0.5} \left\{ 1 + \left[\left(\frac{\mu_i}{\mu_j} \right) \left(\frac{M_j}{M_i} \right)^{0.5} \times \left(\frac{1 + S_i/T}{1 + S_j/T} \right)^{0.5} \right]^2 \frac{1 + S_{ij}/T}{1 + S_i/T} \right\} \quad (32)$$

式中, ϵ_{ij} 为校正系数;

$$\epsilon_{ij} = \epsilon_{ji} = \left[\frac{0.5(M_i + M_j)}{(M_i + M_j)^{0.5}} \right]^{0.25} \quad (33)$$

M_{ij} 为平均分子量;

$$M_{ij} = 0.5(M_i + M_j) \quad (34)$$

S_i, S_j 为组分 i, j 的 Sutherland 常数, 通常 $S = 1.5 T_b$, 对于 H_2, He, Ne 而言, $S = 79$;

S_{ij} 为相互作用 Sutherland 常数,

$$S_{ij} = C_s (S_i S_j)^{0.5} \quad (35)$$

通常 $C_s = 1$, 对含强极性 H_2O, NH_3 气体,

$C_s = 0.733$ 。

高压非极性气体粘度:

$$[(\mu - \mu^0)\xi + 1]^{0.25} = 1.023 + 0.23364\rho_r + 0.58533\rho_r^2 - 0.40758\rho_r^3 + 0.093324\rho_r^4 \quad 0.1 \leq \rho_r < 3 \quad (36)$$

高压极性气体粘度:

$$(\mu - \mu^0)\xi = 1.656\rho_r^{1.111} \quad \rho_r \leq 0.1 \quad (37)$$

$$(\mu - \mu^0)\xi = 0.0607(9.045\rho_r + 0.63)^{1.739} \quad 0.1 < \rho_r \leq 0.9 \quad (38)$$

$$\log\{4 - \log[(\mu - \mu^0)\xi]\} = 0.6349 - 0.1005\rho_r - \Delta \quad 0.9 < \rho_r < 2.6 \quad (39)$$

式中: $\Delta = 0$

$$\Delta = 4.75 \times 10^{-4}(\rho_r^3 - 10.65)^2 \quad 2.2 < \rho_r < 2.6 \quad (40)$$

$$(\mu - \mu^0)\xi = 90 \quad \rho_r = 2.8 \quad (41)$$

$$(\mu - \mu^0)\xi = 250 \quad \rho_r = 3.0 \quad (42)$$

高压气体混合物粘度:

$$(\mu_m - \mu_m^0)\xi_m = 1.08[\exp(1.439\rho_{rm} - \exp(-1.111\rho_{rm}^{1.858}))] \quad (43)$$

混合物假临界参数:

$$T_{cm} = \sum_i Y_i T_{ci} \quad (44)$$

$$Z_{cm} = \sum_i Y_i Z_{ci} \quad (45)$$

$$V_{cm} = \sum_i Y_i V_{ci} \quad (46)$$

$$p_{cm} = Z_{cm} R T_{cm} / V_{cm} \quad (47)$$

$$M_m = \sum_i Y_i M_i \quad (48)$$

$$\rho_{rm} = \rho_m / \rho_{cm} \quad (49)$$

$$\xi_m = T_{cm}^{1/6} M_m^{-1/2} p_{cm}^{-1/2} \quad (50)$$

4. 导热系数

低压气体导热系数(烃类):

$$\lambda = 10^{-6}(14.52T_r - 5.14)^{1/2} C_p / \Gamma \quad (51)$$

$$\text{式中: } \Gamma = T_c^{1/4} M^{1/2} p_c^{-1/2} \quad (52)$$

高压气体导热系数:

$$(\lambda - \lambda^0) \Gamma Z_c^5 = 14.10^{-8} [\exp(0.535\rho_r) - 1] \quad \rho_r \leq 0.5 \quad (53)$$

$$(\lambda - \lambda^0) \Gamma Z_c^5 = 13.1 \times 10^{-8} [\exp(0.67\rho_r) - 1.069] \quad 0.5 < \rho_r < 2.0 \quad (54)$$

$$(\lambda - \lambda^0) \Gamma Z_c^5 = 2.976 \times 10^{-8} [\exp(1.155\rho_r) + 2.016] \quad 2.6 < \rho_r < 2.8 \quad (55)$$

气体混合物导热系数:

$$\lambda_m = \sum_i \frac{Y_i \lambda_i}{\sum_j Y_j A_{ij}} \quad (56)$$

式中: A_{ij} 为结合系数。

$$A_{ij} = \frac{1}{4} \left\{ 1 + \left[\left(\frac{\mu_i}{\mu_j} \right) \left(\frac{M_j}{M_i} \right)^{1/2} \left(\frac{T + S_i}{T + S_j} \right) \right]^2 \right\} \frac{T + S_{ij}}{T + S_i} \quad (57)$$

S_i, S_j, S_{ij} 见式(35)。

结构设计

从具有最大平均温压及最佳传热效果考虑,总是尽可能布置成逆流。多股流换热器的逆流布置如图1所示。

在装置容量较大,板式单元尺寸受钎接炉容量

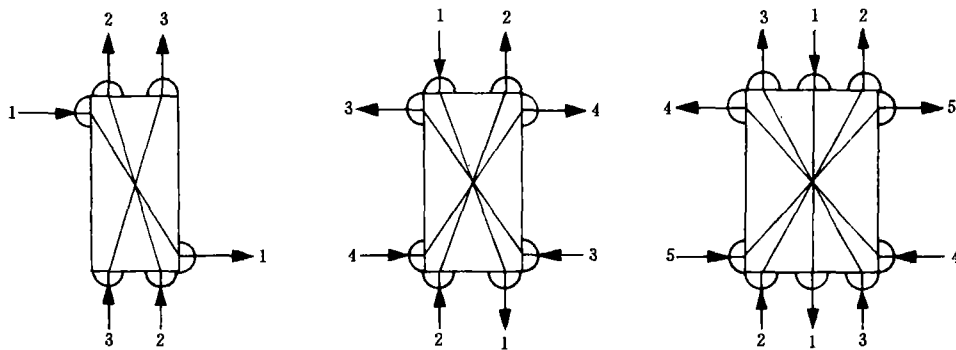


图1 多股流换热器逆流布置的几种型式

Fig. 1. Several counter-current arrangement types of heat exchanger with multistream.

限制的情况下,可用数只板式单元串联、并联、串并联组合。为保证流体均布,推荐用对称式并联布置。见图2。

错流的平均温压小于逆流,一般只用于单侧相变,或返流阻受到严格限制的场合。板翅式换热器通道内的导流片具有如图3所示的几种型式。

上述这些结构型的正确选择应据换热流体的流体动力工况、热力参数、工质热物性而定。正确的设计应使流体在板式单元与通道内均匀分布。板翅式换热器通道的排列组合是结构设计的核心,影响因素复杂,将对换热器的效率、流动阻力及流体均布产生决定性影响。这方面可参考文献[4,5]。

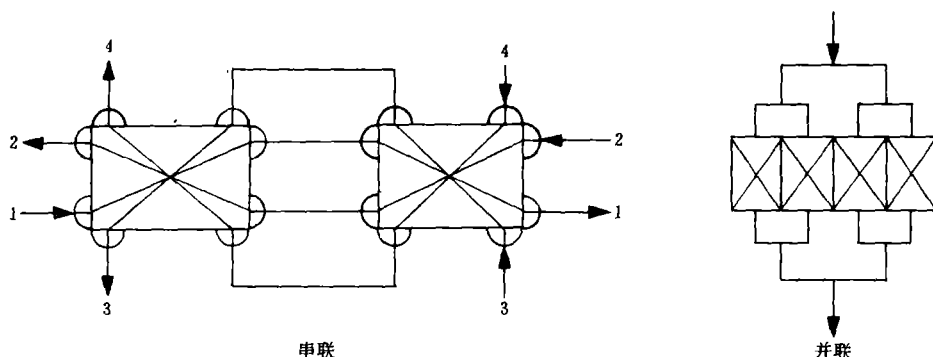


图2 板翅式组合布置

Fig. 2. Plate-fin assembly arrangement.

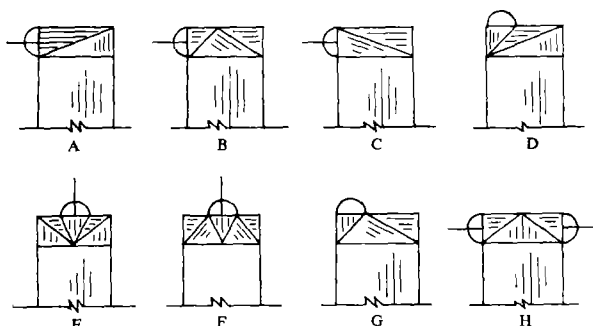


图3 导流片结构型式

Fig. 3. Structural style of flow deflector.

传热计算

多股流传热具有比两股流复杂得多的温度场与换热工况,国外文献虽间有报导,但据其编拟程序进行运算时,或由于模型的缺陷,或受计算方法限制,国内多人尝试过多次都未达到实用目的。这里仍推荐较实用的热组综合分解法。其基本思路是将换热流体按给热、吸热分为两类,综合为相当的两股流,把多股流换热简化为两股流换热。对一股热流体 g 、 n 股冷流体 j ,综合后的热导为:

$$KF = \frac{\alpha_g F_g \eta_g \sum_j \alpha_j F_j \eta_j}{\alpha_g F_g \eta_g + \sum_j \alpha_j F_j \eta_j} \quad (58)$$

换热所需长度为:

$$L = Q / KF \Delta T \quad (59)$$

相应的传热温差按热负荷比例计算。另方面尚要按分解法分析各股冷气流分别与按传热面积比例分配的热气流之间的换热:

$$KF_{j-g} = \frac{\alpha_j F_j \eta_j \alpha_g F_g \eta_g (F_j / \sum_j F_j)}{\alpha_j F_j \eta_j + \alpha_g F_g \eta_g (F_j / \sum_j F_j)} \quad (60)$$

$$L_j = Q_j / KF_{j-g} \Delta T_{j-g} \quad (61)$$

综合与分解计算所得的 L 与 L_j 相对偏差要求少于 0.1。

软件的实用性

为八五攻关及发展板翅式换热器的需要,作者开发、完善了一系列无相变、有相变、多股流换热器设计程序,应用该软件设计、制造了数十台各种规格、参数的换热器。据反馈讯息,无论换热面积、传热温差或流体阻力都达到或超过预期指标。实践证明:上述有关物理模型、设计思路、计算方法的分析是切实可行的。

符号说明

p 为压力; p_{cm} 为假临界压力; p_r 为对比压力; T 为温度; T_{cm} 为假临界温度; T_r 为对比温度; V 为比容; V_{cm} 为假临界比容; V_r 为对比比容; ρ 为密度; ρ_{cm} 为假临界密度; ρ_r 为对比密度; ω 为偏心因子; μ 为动力粘度; C_p 为比热容; λ 为导热系数; T_b 为常压沸点。

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ABSTRACT: Multi-stream heat exchange often occurs in light hydrocarbon processing for gas separation, which is mainly the heat exchange of ordinary untripped gas with multi-stream separated product gas. The physical model, computing method and applied software involved in the computation of heat physical property, structure and heat transfer without phase transition are presented.

SUBJECT HEADINGS: Light hydrocarbon, Processing, Gas flow, Heat exchanger, Physical property, Structure design, Heat transfer, Computation.

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Zhang Yufang (*Southwest Petroleum Institute*): **TECHNOLOGICAL PROCESS OF RECOVERING LIGHT ENDS FROM LOW PRESSURE GASES**, NGI 16(1), 1996: 72~76

ABSTRACT: Low pressure gas often refers to associated gas and stabilized end gas in crude oil. They are characterized by low pressure and abundant light ends. Extracting light ends from natural gas is able to enhance the quality of commercial gas and the overall economic benefit. The technical process of recovering light ends from natural gas by low temperature separation is discussed. It is a very economical selection for low pressure gas adopting composite refrigeration process, combining external cold source cyclic refrigeration with gas expansion refrigeration, to recover light ends from natural gas after computing and comparing several recovery programs.

SUBJECT HEADINGS: Low pressure, Natural gas, Light ends recovery, Method, Selection.

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Xiang Zhineng (*Planning Department of Sichuan Petroleum Administration*): **DRAWING ON THE OVERSEAS EXPERIENCE TO ENHANCE INVESTMENT MANAGEMENT OF OIL ENTERPRISES**, NGI 16(1), 1996: 77~79

ABSTRACT: The experiences of western oil companies in the aspects of investment and management are introduced. It is pointed out that the investment management of China's oil-gas enterprises must overall set up the conception of market and benefit, strengthen the research on enterprise development strategy, consummate the system of investment responsibility, do the comprehensive balance of investment well and improve the project management, to enhance the economic benefit in investment.

SUBJECT HEADINGS: Oil enterprise, Investment, Management, Reform, Investment benefit.

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Tang Tingchuan (*Technology and Economy Research Institute of Natural Gas Industry of Sichuan Petroleum Administration*): **PRESENT SITUATION, PROBLEMS AND MEASURES OF FOSTERING CHINA'S GAS MARKET SYSTEM**, NGI 16(1), 1996: 80~83

ABSTRACT: It is an inexorable selection of reforming natural gas industry and raising efficiency to foster and develop gas market system in the market economy. Gas market system consists of exploration and development market, specialized technical operation market and other element markets such as technology, funds and labour force etc. and the selling market. All these markets have not developed completely except selling market. Aimed at the problems existing in gas market development, a series of measures are adopted, such as setting up a new conception in accordance with the market economy, creating the main body of gas market, promoting the commercialization of objective market, consummating the organization of gas market and doing well the basic jobs of various links of gas market.

SUBJECT HEADINGS: Natural gas, Market system, Measures, Development, Basic jobs.

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